W'ALL IN ONE: MODA

Modeling on an envelope scale

			OVERVIEW of the simulation		
1	User Case	performance of systems on 11	Heat and moisture transfer modeling on system (wall envelope) level to assess the thermal and moist performance of such systems in different boundary conditions, to analyze the impact of the newly developed ystems on 1D heat losses, on thermal bridge (2D) heat losses, and on inner surface temperature with condensation and mould growth risk assessment.		
2	CHAIN OF MODELS	Model 1	Heat transport: continuum model conservation of energy equation (1st law of thermodynamics) • Heat transfer due to conduction (diffusion) • Heat transfer due to convection • Heat transfer due to radiation • Fourier's law (conductive flow) • Newton's law of cooling (convective flow) • Stefan Boltzmann's law (radiative flow)		
		Model 2	Moisture transport: continuum model conservation of mass (moisture transfer) • Fick's first law (vapour flow) • Darcy's law (liquid flow)		
3	PUBLICATION ON THE SIMULATION				
4	ACCESS	bought softwa	re license (for WUFI or Delphin)		

1	ASPECT OF THE USER CASE/SYSTEM TO BE SIMULATED		
	ASPECT OF THE USER	Analyze the impact of adding the newly developed system at the interior surface of historical	
	CASE TO BE	brick walls (taken as an e.g.) on the heat losses, and on inner surface temperature with a	
1.1	SIMULATED	condensation and mould growth risk assessment	
1.1	AND HOW IT FORMS		
	A PART OF THE TOTAL		
	USER CASE		
1.2	Material	e.g.: Brick with Aerogel blanket layer and plaster on the interior	
1.3	GEOMETRY	Cartesian (1D)	
1.4	TIME LAPSE	5 years	
	MANUFACTURING	External forces: heat (for e.g. solar radiation) and moisture (for e.g. rain)	
1.5	PROCESS OR IN-	(the source is the weather data)	
	SERVICE CONDITIONS		
	PUBLICATION		
1.6	ON THIS ONE		
	SIMULATION		

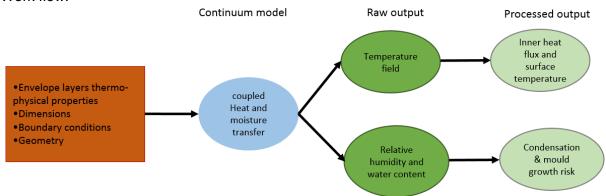
2	GENERIC PHY	ICS OF THE MODEL EQUATION		
2.0	MODEL TYPE AND NAME	Tightly coupled equations: Heat transport: continuum model Moisture transport: continuum model		
2.1	MODEL ENTITY	finite volumes		
2.2	Model Physics/	Equations $PE1: \frac{dH}{dT} \frac{\partial T}{\partial t} = \nabla(\lambda \nabla T) + h_v \nabla(\delta_p \nabla(\varphi P_{sat}))$		

	CHEMISTRY EQUATION PE'S		PE2: $\frac{dw}{d\varphi} \frac{\partial \varphi}{\partial t} = \nabla (D_{\varphi} \nabla \varphi + \delta_{p} \nabla (\varphi P_{sat}))$
		Physical quantities for each equation	PE1: Physical quantities H=enthalpy (J) $T = \text{temperature } (K)$ $\varphi = \text{relative humidity } (-)$ $\lambda = \text{thermal conductivity } (W/(mK))$ $t = \text{time } (s)$ $\delta_p = \text{water vapour permeability } (kg/(msPa))$ $hv = \text{evaporation enthalpy } (J/kg)$ $Psat = \text{water vapour saturation pressure } (Pa)$
			PE2: Physical quantities $w = \text{water content (kg/m3)}$ $\varphi = \text{relative humidity (-)}$ $D_{\varphi} = \text{liquid conduction coefficient (kg/(ms))}$ $t = \text{time (s)}$ $\delta_p = \text{water vapour permeability (kg/(msPa))}$ $Psat = \text{water vapour saturation pressure (Pa)}$
2.3. M RELATIO	ATERIALS DNS	 The mater The mater water vape the materi The sorpti 	rial's thermal conductivity as a function of temperature (for PE1) rial's thermal conductivity as a function of moisture content (for PE1) rial's specific heat capacity (for PE1) our permeability or water vapour diffusion resistance factor (for PE1 and PE2) al's liquid conduction coefficient or liquid absorption coefficient (for PE2) on isotherm curve (for PE2) re are the results of experimental measurments)
2.4	SIMULATED INPUT	NA	

3	SPECIFIC COMPUT	ATIONAL MOD	DELLING METADATA
3.1	Numerical Solver	Finite volume • Fully	implicit for the discretization in time
3.2	SOFTWARE TOOL		nine (https://wufi.de/en/; http://bauklimatik ohin/index.php?aLa=en)
3.3	TIME STEP	Variable (1h or	less)
3.4	COMPUTATIONAL REPRESENTATION	PHYSICS EQUATION, MATERIAL RELATIONS, MATERIAL	written up for finite volumes
		BOUNDARY CONDITIONS	The physics b.c. are written up for the faces of the outer finite volumes
		ADDITIONAL SOLVER	
		PARAMETERS	

4	POST PROCESSING		
	THE PROCESSED	Temperatures will be used to estimate the thermal losses/gains throughout the envelope	
4.1	OUTPUT IS CALCULATED	Relative humidity will be used to estimate the moisture risks (such as condensation)	
	FOR		
4.2	METHODOLOGIES	physics definition for loss and gain	
4.2		literature relations for condensations and mould as function of temperature and humidity	
4.3	Margin Of Error		

Work flow:



Modeling on a building scale

		OVERVIEW of the simulation		
1	User Case	Assess the impact of applying the developed internal thermal insulation systems to retrofit an old nouse on the energy demands and thermal comfort.		
2	CHAIN OF MODELS	Heat transport: continuum model conservation of energy equation (1st law of thermodynamics) • Heat transfer due to conduction (diffusion) • Heat transfer due to convection • Heat transfer due to radiation • Fourier's law (conductive flow) • Newton's law of cooling (convective flow) • Stefan Boltzmann's law (radiative flow)		
3	PUBLICATION ON THE SIMULATION			
4	Access CONDITIONS	Free software (for EnergyPlus)		

	Acres of The Hea	CACE (SYCTEM TO DE SIMULATED		
1	ASPECT OF THE USE	ASPECT OF THE USER CASE/SYSTEM TO BE SIMULATED		
	ASPECT OF THE USER	analyze the impact of adding the newly developed system at the interior surface of to retrofit		
	CASE TO BE	an old house on the heating and/or cooling demands		
1.1	SIMULATED			
1.1	AND HOW IT FORMS			
	A PART OF THE TOTAL			
	USER CASE			
1.2	Material	e.g.: an old brick house with Aerogel blanket layer on the interior		
1.3	GEOMETRY	Cartesian		
1.4	TIME LAPSE	1 year		
	MANUFACTURING	Statistical weather conditions		
1.5	PROCESS OR IN-			
	SERVICE CONDITIONS			
1.6	Publication			
	ON THIS ONE			
	SIMULATION			

2	GENERIC PHY	SICS OF THE MODEL EQUATION
2.0	MODEL TYPE	Heat transport: continuum model
2.0	AND NAME	
2.1	Model	Finite volume

	ENTITY		
	MODEL PHYSICS/ CHEMISTRY EQUATION PE'S	Equations	PE1: $\rho cp \frac{\partial T}{\partial t} = \nabla(\lambda \nabla T)$
2.2		Physical quantities for each equation	PE1: Physical quantities $T = \text{temperature } (K)$ $\lambda = \text{thermal conductivity } (W/(mK))$ $t = \text{time } (s)$ $\rho = \text{density } (kg/m3)$ $cp = \text{specific heat } (J/(kg.K))$
2.3. M	ATERIALS DNS	2. The	material's thermal conductivity (for PE1) material's specific heat capacity and density (for PE1) e are the results of experimental measurements)
2.4	SIMULATED INPUT	1	

3	SPECIFIC COMPUT	ATIONAL MOD	ELLING METADATA		
3.1	NUMERICAL SOLVER		Finite difference Difference scheme: FullyImplicitFirstOrder		
3.2	SOFTWARE TOOL	EnergyPlus (htt	tps://energyplus.net/)		
3.3	TIME STEP	1h or less			
3.4	COMPUTATIONAL REPRESENTATION	PHYSICS EQUATION, MATERIAL RELATIONS, MATERIAL	written up for finite difference		
		BOUNDARY CONDITIONS	Weather data (temperature, radiation,)		
		ADDITIONAL SOLVER PARAMETERS	Inside face temperature convergence criteria: <0.0001		

4	POST PROCESSING		
	THE PROCESSED	Temperatures will be used to estimate heating/cooling loads	
4.1	OUTPUT IS CALCULATED		
	FOR		
4.2	METHODOLOGIES	physics definition for loss and gain	
4.3	Margin Of Error		